SAE 8620 Case Core Composite Steel Iteration #70

MICROSTRUCTURAL DATA, MONOTONIC AND FATIGUE TEST RESULTS

F. Yin and A. Fatemi

Department of Mechanical, Industrial, and Manufacturing Engineering The University of Toledo Toledo, OH 43606

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NOMENCLATURE

A_o, A_f	initial, final area	S	engineering stress
HB, HRB, HRC	Brinell, Rockwell B-Scale, Rockwell C-Scale hardness number	YS, UYS, LYS, YS'	monotonic yield, upper yield, lower yield, cyclic yield strength
b, c, n	fatigue strength, fatigue ductility, strain hardening exponent	YPE	yield point elongation
D _o , D _f	initial, final diameter	$\mathbf{S}_{\mathbf{u}}$	ultimate tensile strength
e	engineering strain	%EL	percent elongation
E, E'	monotonic, midlife cycle modulus of elasticity	%RA	percent reduction in area
K, K'	monotonic, cyclic strength coefficient	$\sigma, \sigma_{f}, \sigma_{f}$	true stress, true fracture strength, fatigue strength coefficient
L _o , L _f	initial, final gage length	$\sigma_a, \sigma_m, \Delta \sigma$	stress amplitude, mean stress, stress range
$N_{50\%}, (N_f)_{10\%},$ $(N_f)_{50\%},$	number of cycles to midlife, 10% load drop, 50% load drop	$\epsilon_{e}, \epsilon_{p}, \epsilon$	true elastic, plastic, total strain
2N _f	reversals to failure	$\epsilon_{\rm f}, \epsilon_{\rm f}$	true fracture ductility, fatigue ductility coefficient
P_f, P_u	fracture, ultimate load	$\varepsilon_{a}, \varepsilon_{m}, \Delta \varepsilon$	strain amplitude, mean strain, strain range
R	neck radius; or strain ratio	$\Delta \varepsilon_{e}, \Delta \varepsilon_{p}$	elastic, plastic strain range

UNIT CONVERSION TABLE

Measure	SI Unit US Unit		from SI to US	from US to SI
Length	mm	in	1 mm = 0.03937 in 1 mm2 = 0.00155 in2 1kN = 0.2248 klb 1 MPa = 0.14503 ksi °C = (°F - 32)/1.8	1 in = 25.4 mm
Area	mm ²	in ²		1 in ² = 645.16 mm ²
Load	kN	klb		1 klb = 4.448 kN
Stress	MPa	ksi		1 ksi = 6.895 MPa
Temperature	°C	°F		${}^{\circ}F = ({}^{\circ}C * 1.8) + 32$
<u>In SI Unit</u> :	$1 \text{ kN} = 10^3 \text{ N}$	1 Pa = 1 N/m ²	1 MPa = 10^{6} Pa = 1 N/mm ²	1 Gpa = 10 ⁹ Pa
<u>In US Unit</u> :	$1 \text{ klb} = 10^3 \text{ lb}$	1 psi = 1 lb/in ²	1 ksi = 10^{3} psi	

SUMMARY

The monotonic properties, and fatigue behavior data have been obtained for SAE 8620 Case-Core Composite steel. The material was provided by MacSteel Company. Two tensile tests were performed to acquire the desired monotonic properties. Eighteen fatigue tests were performed to obtain the strain-life and cyclic stress-strain curves and properties. The experimental procedure followed and results obtained are presented and discussed in this report.

I. EXPERIMENTAL PROGRAM

1.1 Material and Specimen Fabrication

1.1.1 Material

The SAE 8620 Case Core steel was provided by MacSteel Company. This material was delivered to the University of Toledo in round bar form. The bars were approximately 1 inch in diameter. In Table 1, the chemical composition supplied by AISI is shown. The hardness profile is shown in Figure 1b.

1.1.2 Specimen

In this study, identical round specimens were used for the monotonic and fatigue tests. The specimen configuration and dimensions are shown in Figure 1a. This configuration deviates slightly from the specimens recommended by ASTM Standard E606 [1]. The recommended specimens have uniform or hourglass test sections. The specimen geometry shown in Figure 1a differs by using a large secondary radius throughout the test section.

All specimens were machined in the Mechanical, Industrial, and Manufacturing Engineering Machine Shop at the University of Toledo. The specimens were cut to the appropriate length, after that center-drilled in both ends and inserted into a CNC machine. Using the CNC machine, specimens were rough machined. They were then heat treated and ground.

A commercial round-specimen polishing machine was used to polish the specimen gage section. Three different grits of aluminum oxide lapping film were used: 15μ , 9μ , and 3μ .

The 3μ grit was used as the final polish and polishing marks coincided with the specimens' longitudinal direction. The polished surfaces were carefully examined under magnification to ensure complete removal of machine marks within the test section.

1.2 Testing Equipment

1.2.1 Apparatus

An MTS closed-loop servo-hydraulic axial load frame in conjunction with an Instron Fast-Track digital servo-controller was used to conduct the tests. The calibration of this system was verified prior to beginning the test program. The load cell used had a capacity of 22 klb. Hydraulically operated Wedge grips were employed to secure the specimens' ends in series with the load cell.

Total strain was controlled for all tests using an extensometer rated as ASTM class B1 [2]. The calibration of the extensometer was verified using displacement apparatus containing a micrometer barrel in divisions of 0.0001 in. The extensometer had a gage length of 0.30 in and was capable of measuring strains up to 15 %.

In order to protect the specimens' surface from the knife-edges of the extensometer, ASTM Standard E606 recommends the use of transparent tape or epoxy to 'cushion' the attachment. For this study, it was found that application of transparent tape strips was difficult due to the radius within the test section. Therefore, epoxy was considered to be the best protection. One disadvantage of epoxy is the variability of mixtures throughout the test program. As an alternative to epoxy, M-coat D offered a more consistent mixture. Therefore, the tests were performed using M-coat D.

All tests were conducted at room temperature and were monitored using a digital thermometer. In order to minimize temperature effects upon the extensometer and load cell calibrations, fluctuations were maintained within ± 2 °C (± 3.6 °F) as required by ASTM Standard E606. Also, the relative humidity of the air was monitored using a precision hydrometer.

1.2.2 Alignment

Significant effort was put forth to align the load train (load cell, grips, specimen, and actuator). Misalignment can result from both tilt and offset between the central lines of the load train components. According to ASTM Standard E606, the maximum bending strains should not exceed 5 % of the minimum axial strain range imposed during any test program. For this study, the minimum axial strain range was 0.0065 in/in. Therefore, the maximum allowable bending strain was 325 microstrain. ASTM Standard E1012, Type A, Method 1 was followed to verify specimen alignment [3]. For this procedure, two arrays of four strain gages per array were arranged at the upper and lower ends of the uniform gage section. For each array, gages were equally spaced around the circumference of a 0.5-in. uniform diameter bar. The maximum bending strain determined from the gaged specimen was less than 60 microstrain. This value was well within the allowable ASTM limit.

1.3 Test Methods and Procedures

1.3.1 Monotonic tension tests

All monotonic tests in this study were performed using test methods specified by ASTM Standard E8 [4]. Two specimens were used to obtain the monotonic properties. To protect the extensometer, strain control was used initially and then displacement control was used until fracture.

For the elastic and initial yield region (0% to 0.5% strain), a strain rate of 0.0025 in/in/min was chosen. This strain rate was three-quarters of the maximum allowable rate specified by ASTM Standard E8 for the initial yield region. After yielding (0.5% strain to fracture), the strain rate was increased by a factor of three (i.e., 0.0075 in/in/min). After the extensometer was removed, a displacement rate of 0.1275 in/min was used. This displacement rate provided approximately the same strain rate as that used prior to switching control modes.

After the tension tests were concluded, the broken specimens were carefully reassembled. The final gage lengths of the fractured specimens were measured with a Vernier caliper having divisions of 0.001 in. Using an optical comparator with 10X magnification and divisions of 0.001 in, the final diameter and the neck radius were measured. It should be noted that prior to the test, the initial minimum diameter was measured with this same instrument.

1.3.2 Constant amplitude fatigue tests

All constant amplitude fatigue tests in this study were performed according to ASTM Standard E606. It is recommended by this standard that at least 10 specimens be used to generate the fatigue properties.

For this study, 18 specimens at 7 different strain amplitudes ranging from 0.325% to 2% were utilized. Instron LCF software was used in all tests, except for tests done in load control and displacement control (in which Instron SAX software was used after changing to load control mode). During each test, the total strain was recorded using the extensometer output. Test data were automatically recorded throughout each test.

There were three control modes used for these tests. Strain control was used in the tests with plastic deformation (2%, 1.5%, 1%, 0.65% and 0.5% strain amplitudes). For five tests at 0.65% and 0.5% strain amplitude, strain control was used initially and load control was used for the remainder of the tests to prevent high mean stress. Displacement control was used in three of the higher level tests (1% and 0.65% strain amplitudes) to free the specimen surface inside the gage length, so replicas could be taken to detect any short cracks during fatigue tests. First, displacement amplitude was monitored in these tests during strain control, then the same displacement amplitude was used in the displacement control test. For all the elastic tests (0.4% and 0.325% strain amplitudes) load control was used although strain control was used initially in one of the tests to determine the stabilized load, then load control was used for the remainder of the tests. For the tests starting with strain control, the applied frequencies ranged from 0.1 Hz to 2 Hz in order to keep a strain rate about 0.02 in/in/sec. For the load control tests, the frequency was increased between 2 Hz and 25 Hz in order to shorten the overall test duration. All strain control tests were conducted using a triangular waveform.

II. EXPERIMENTAL RESULTS AND ANALYSIS

2.1 Microstructural Data

Photomicrographs of the microstructure were obtained using an optical microscope with a digital camera attachment. In Figures 2a and 2b, the transverse and longitudinal direction are both shown for case, core and transition area at 500X magnification. It can be seen from this photomicrograph that SAE 8620 Case-Core composite steel had a martens ite microstructure. In Figures 3a and 3b, the inclusions/voids in T'-T direction and L-T direction are shown at 100X magnification. For Figures 2a and 3a, the rolling direction is perpendicular to the page. For Figures 2b and 3b, the rolling direction is horizontal to the page.

According to ASTM Standard E45, method A, the inclusion rating number for type A inclusion in T'-T direction and L-T direction was found [6]. Residual stresses were also measured. The residual stress profile is shown in Figure A.15 (superimposed with the residual stress profile of It_62). The hardness profile is shown in Figure 1b. A summary of the microstructural data for SAE 8620 Case-Core Composite steel is provided in Table 2.

2.2 Monotonic Deformation Behavior

The properties determined from monotonic tests were the following: modulus of elasticity (E), yield strength (YS), ultimate tensile strength (S_u), percent elongation (%EL), percent reduction in area (%RA), true fracture strength (? f), true fracture ductility (? f), strength coefficient (K), and strain hardening exponent (n).

True stress (σ), true strain (ϵ), and true plastic strain (ϵ_p) were calculated from engineering stress (S) and engineering strain (e), according to the following relationships which are based on constant volume assumption:

$$\boldsymbol{s} = S(1+e) \tag{1a}$$

$$\boldsymbol{e} = \ln(1+\boldsymbol{e}) \tag{1b}$$

$$\boldsymbol{e}_{p} = \boldsymbol{e} - \boldsymbol{e}_{e} = \boldsymbol{e} - \frac{\boldsymbol{S}}{E}$$
(1c)

The true stress (σ) - true strain (ε) plot is often represented by the Ramberg-Osgood equation:

$$\boldsymbol{e} = \boldsymbol{e}_{e} + \boldsymbol{e}_{p} = \frac{\boldsymbol{s}}{E} + \left(\frac{\boldsymbol{s}}{K}\right)^{\frac{1}{n}}$$
(2)

The strength coefficient, K, and strain hardening exponent, n, are the intercept and slope of the best line fit to true stress (σ) versus true plastic strain (ϵ_p) data in log-log scale:

$$\boldsymbol{s} = K \left(\boldsymbol{e}_{p} \right)^{n} \tag{3}$$

In accordance with ASTM Standard E739 [8], when performing the least squares fit, the true plastic strain (ε_p) was the independent variable and the stress (σ) was the dependent variable. These plots for the two tests conducted are shown in Figure 4. To generate the K and n values, the range of data used in this figure was chosen according to the definition of discontinuous yielding specified in ASTM Standard E646 [9]. Therefore, the valid data range occurred between the end of yield point extension and the strain at or prior to maximum load.

The true fracture strength, σ_f , was corrected for necking according to the Bridgman correction factor [9]:

$$\boldsymbol{s}_{f} = \frac{\frac{P_{f}}{A_{f}}}{\left(1 + \frac{4R}{D_{f}}\right) \ln\left(1 + \frac{D_{f}}{4R}\right)}$$
(4)

where P_f is the load at fracture, R is the neck radius, and D_f is the diameter at fracture. The true fracture ductility, ε_f , was calculated from the relationship based on constant volume:

$$\boldsymbol{e}_{f} = \ln\left(\frac{A_{o}}{A_{f}}\right) = \ln\left(\frac{1}{1-RA}\right)$$
(5)

where A_f is the cross-sectional area at fracture, A_o is the original cross-sectional area, and RA is the reduction in area.

A summary of the monotonic properties for SAE 8620 Case Core Composite steel is provided in Table 2. The monotonic stress-strain curves for two tests are shown in Figure 5. As can be seen from this figure, the two curves for the composite case-core material are very close to each other. Refer to Table A.1 in the Appendix for a summary of the monotonic test results.

2.3 Cyclic Deformation Behavior

2.3.1 Transient cyclic response

Transient cyclic response describes the process of cyclic-induced change in deformation resistance of a material. Data obtained from constant amplitude strain-controlled fatigue tests were used to determine this response. Plots of stress amplitude variation versus applied number of cycles can indicate the degree of transient cyclic softening/hardening. Also, these plots show when cyclic stabilization occurs.

A composite plot of the transient cyclic response for SAE 8620 Case-Core Composite steel is shown in Figure A.1 of the Appendix. The transient response is normalized on the rectangular plot in Figure A.1a, while a semi-log plot is shown in Figure A.1b. Even though multiple tests were conducted at each strain amplitude, data from one test at each strain amplitude tested are shown in these plots.

2.3.2 Steady-state cyclic deformation

Another cyclic behavior of interest was the steady state or stable response. Data obtained from constant amplitude strain-controlled fatigue tests were also used to determine this response. The properties determined from the steady-state hysteresis loops were the following: cyclic modulus of elasticity (E'), cyclic strength coefficient (K'), cyclic strain hardening exponent (n'), and cyclic yield strength (YS'). Half-life (midlife) hysteresis loops and data were used to obtain the stable cyclic properties.

Similar to monotonic behavior, the cyclic true stress-strain behavior can be characterized by the Ramberg-Osgood type equation:

$$\frac{\Delta \boldsymbol{e}}{2} = \frac{\Delta \boldsymbol{e}_{e}}{2} + \frac{\Delta \boldsymbol{e}_{p}}{2} = \frac{\Delta \boldsymbol{s}}{2 E} + \left(\frac{\Delta \boldsymbol{s}}{2 K}\right)^{\frac{1}{n}}$$
(6)

It should be noted that in Equation 6 and the other equations that follow, E is the average modulus of elasticity that was calculated from the monotonic tests.

The cyclic strength coefficient, K', and cyclic strain hardening exponent, n', are the intercept and slope of the best line fit to true stress amplitude ($\Delta\sigma/2$) versus true plastic strain amplitude ($\Delta\varepsilon_p/2$) data in log-log scale:

$$\frac{\Delta s}{2} = K \left(\frac{\Delta e_p}{2} \right)^n \tag{7}$$

In accordance with ASTM Standard E739, when performing the least squares fit, the true plastic strain amplitude ($\Delta \varepsilon_p/2$) was the independent variable and the stress amplitude ($\Delta \sigma/2$) was the dependent variable. The true plastic strain amplitude was calculated by the following equation:

$$\frac{\Delta \boldsymbol{e}_{p}}{2} = \frac{\Delta \boldsymbol{e}}{2} - \frac{\Delta \boldsymbol{s}}{2E}$$
(8)

This plot is shown in Figure 6. To generate the K' and n' values, the range of data used in the figure was chosen for $[\Delta \varepsilon_p/2]_{calculated} \ge 0.00018$ in/in.

The cyclic stress - strain curve reflects the resistance of a material to cyclic deformation and can be vastly different from the monotonic stress - strain curve. The cyclic stress - strain curve is shown in Figure 7. In Figure 8, superimposed plots of monotonic and cyclic curves are shown. As can be seen in Figure 8, SAE 8620 Case-Core Composite steel cyclically softens. Figure A.2 in the Appendix shows a composite plot of the steady-state (midlife) hysteresis loops. Even though multiple tests were conducted at each strain amplitude, the stable loops from only one test at each strain amplitude are shown in this plot.

2.4 Constant Amplitude Fatigue Behavior

Constant amplitude strain-controlled fatigue tests were performed to determine the strainlife curve. The following equation relates the true strain amplitude to the fatigue life:

$$\frac{\Delta \mathbf{e}}{2} = \frac{\Delta \mathbf{e}_{e}}{2} + \frac{\Delta \mathbf{e}_{p}}{2} = \frac{\mathbf{s}_{f}}{E} \left(2 N_{f} \right)^{b} + \mathbf{e}_{f} \left(2 N_{f} \right)^{c}$$
(9)

where σ_{f} is the fatigue strength coefficient, b is the fatigue strength exponent, ϵ_{f} is the fatigue ductility coefficient, c is the fatigue ductility exponent, E is the monotonic modulus of elasticity, and $2N_{f}$ is the number of reversals to failure (which was defined as a 50% load drop, as recommended by ASTM Standard E606).

The fatigue strength coefficient, σ_f , and fatigue strength exponent, b, are the intercept and slope of the best line fit to true stress amplitude ($\Delta\sigma/2$) versus reversals to failure (2N_f) data in log-log scale:

$$\frac{\Delta \boldsymbol{s}}{2} = \boldsymbol{s}_{f} \left(2 N_{f} \right)^{b} \tag{10}$$

In accordance with ASTM Standard E739, when performing the least squares fit, the stress amplitude ($\Delta\sigma/2$) was the independent variable and the reversals to failure (2N_f) was the dependent variable. This plot is shown in Figure 9. To generate the σ_f ' and b values, the range of data used in this figure was chosen for $10^2 < N_f \le 10^6$ cycles.

The fatigue ductility coefficient, ε_{f} , and fatigue ductility exponent, c, are the intercept and slope of the best line fit to calculated true plastic strain amplitude ($\Delta \varepsilon_{p}/2$) versus reversals to failure (2N_f) data in log-log scale:

$$\left(\frac{\Delta \boldsymbol{e}_p}{2}\right)_{calculated} = \boldsymbol{e}_f' \left(2N_f\right)^C \tag{11}$$

In accordance with ASTM Standard E739, when performing the least squares fit, the calculated true plastic strain amplitude ($\Delta \varepsilon_p/2$) was the independent variable and the reversals to failure (2N_f) was the dependent variable.

The calculated true plastic strain amplitude was determined from Equation 8. This plot is shown in Figure 10. To generate the ε_{f} and c values, the range of data used in this figure was chosen for $[\Delta \varepsilon_{p}/2]_{calculated} \ge 0.00018$ in/in. Acetate replicas were taken inside the gage length of the specimen surface every 10% of failure life for two fatigue tests (at 1% and 0.65% strain amplitudes) conducted by displacement control. Acetate replicas were also taken for the broken samples of tests at 2% and 1.5% strain amplitudes. No evidence of short cracks was found in the replicas.

The true strain amplitude versus reversals to failure plot is shown in Figure 11. Tests at 2% and 1.5% strain amplitudes were not included in the fittings in Figures 9, 10 and 11 because they were suspected to have a different kind of failure mode. This plot displays the strain - life curve (Eqn. 9), the elastic strain portion (Eqn. 10), the plastic strain portion (Eqn. 11), and superimposed fatigue data. Subsurface failure occurred for tests where $e_a = 0.4\%$, as indicated in Table A.2, and a typical photo is shown in Figure A.14. A summary of the cyclic properties for SAE 8620 Case-Core Composite steel is provided in Table 2. Table A.2 in the Appendix provides the summary of the fatigue test results. Superimposed curves with case, core and casecore composite are shown in Figure A.4 to Figure A.8, from which it can be clearly seen that all of the case-core composite curves generally lie in between the case and core curves, as expected. Superimposed curves with two 8620 Composites (Iter_62 and Iter_70) are shown in Figure A.9 to Figure A.13, from which it can be seen that the two behaviors are close to each other. Superimposed hardness profiles of two 8620 Composites (Iter_62 and Iter_70) are shown in Figure A.16. Pictures of fracture surface are shown in Figure A.17a to Figure A.17f. Some obvious rings can be seen at the edge of the fracture surface of some surface failure pictures (as marked in the pictures). 13

Element	<u>Wt. %</u>
Carbon, C	0.20%
Manganese, Mn	0.96%
Phosphorus, P	0.008%
Sulfur, S	0.030%
Silicon, Si	0.24%
Nickel, Ni	0.53%
Chromium, Cr	0.55%
Molybdenum, Mo	0.22%
Copper, Cu	0.15%
Tin, Sn	0.007%
Aluminum, Al	0.02%
Calcium, Ca	0.001%
Nitrogen, N	0.0088%

Table 1: Chemical composition of SAE 8620 steel

Table 2: Summary of the Mechanical Properties

Microstructural Data	Ave	erage				
ASTM grain size number (MAG=1000X):						
The longitudinal direction (L-T)	5	to 6				
The transverse direction (T'-T)	5	to 6				
Inclusion rating number (MAG=100X):						
Type A (sulfide type), thin series	1 t	o 1.5				
Type B (alumina type), thin series	N	lone				
Type C (silicate type), thin series	N	lone				
Type D (globular type), thin series	N	lone				
Hardness:						
Brinell (HB)						
Transverse direction (T-T')	1	NA				
The first longitudinal direction (L-T)	1	NA				
Rockwell B-scale (HRB)						
Transverse direction (T-T')	I	NA				
The first longitudinal direction (L-T)	I	NA				
Rockwell C-scale (HRC)						
Transverse direction (T-T')	See Hardness	Profile (Fig. 1b)				
The first longitudinal direction (L-T)	I	NA				
Microstructure type:						
Transverse direction (T-T')	mart	tensitic				
Monotonic Properties	Ave	erage	Range			
Modulus of elasticity, E, GPa (ksi):	206.5	(29,943.7)	206.4 - 206.5	(29,935.0 - 29,952.3)		
Yield strength (0.2% offset), YS, MPa (ksi):	1401.7	(203.3)	1390.8 - 1412.5	(201.7 - 204.9)		
Upper yield strength UYS, MPa (ksi):	NA					
Lower yield strength LYS, MPa (ksi):	NA					
Yield point elongation, YPE (%):	NA					
Ultimate strength, S _u , MPa (ksi):	1677.2	(243.3)				
Percent elongation, %EL (%):	10.7%					
Percent reduction in area, %RA (%):	14.2%					
Strength coefficient, K, MPa (ksi):	2,524.1	(366.1)				
Strain hardening exponent, n:	0.0923					
True fracture strength, σ_f^* , MPa (ksi):	1851.7	(268.6)				
True fracture ductility, ϵ_{f} (%):	15.3%					
Cyclic Properties	Av	erage	F	Range		
Cyclic modulus of elasticity, E', GPa (ksi):	199.6	(28,944.7)	183.5 - 209.6	(26,609.6) - (30,404.2)		
Fatigue strength coefficient, σ_{f} , MPa (ksi):	2,053.5	(297.8)				
Fatigue strength exponent, b:	-0.0712					
Fatigue ductility coefficient, ε_{f} :	0.5173					
Fatigue ductility exponent, c:	-0.7397					
Cyclic yield strength, YS', MPa (ksi)	1266.2	(183.6)				
Cyclic strength coefficient, K', MPa (ksi):	2,886.4	(418.6)				
Cyclic strain hardening exponent, n':	0.1326	· /				
Fatigue strength @ 10 ⁶ cycles. S. Mpa (ksi)	731.0	(106.0)				
	, 51.0	(100.0)				

*Correction was made according to Bridgman correction factor.



Figure 1a: Specimen configuration and dimensions



Figure 1b: Hardness profile







b: transition

c: case

Figure 2a: Photomicrograph in the transverse direction (T'-T) at 500X for SAE 8620 Case-Core Composite steel (rolling direction is perpendicular to the page)

20 µm





b: transition

c: case

Figure 2b: Photomicrograph in the longitudinal direction (L-T) at 500X for SAE 8620 Case-Core Composite steel (rolling direction is horizontal to the page)

20 µm







b: transition

c: case

Figure 3a: Examples of inclusions in the transverse direction (T'-T) at 100X for SAE 8620 Case-Core Composite steel (rolling direction is perpendicular to the page)





b: transition

c: case

Figure 3b: Examples of inclusions in the longitudinal direction (L-T) at 100X for SAE 8620 Case-Core Composite steel (rolling direction is horizontal to the page)



Figure 4: True stress versus true plastic strain



Figure 5: Monotonic stress-strain curve



Figure 6: True stress amplitude versus calculated true plastic strain amplitude



Figure 7: True stress amplitude versus true strain amplitude



Figure 8: Composite plot of cyclic and monotonic stress-strain curves



Figure 9: True stress amplitude versus reversals to failure



Figure 10: Calculated true plastic strain amplitude versus reversals to failure



Figure 11: True strain amplitude versus reversals to failure

REFERENCES

- [1] ASTM Standard E606-92, "Standard Practice for Strain-Controlled Fatigue Testing," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 523-537.
- [2] ASTM Standard E83-96, "Standard Practice for Verification and Classification of Extensometers," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 198-206.
- [3] ASTM Standard E1012-93a, "Standard Practice for Verification of Specimen Alignment Under Tensile Loading," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 699-706.
- [4] ASTM Standard E8-96a, "Standard Test Methods for Tension Testing of Metallic Materials," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 56-76..
- [5] ASTM Standard E112-96, "Standard Test Methods for Determining Average Grain Size," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 227-249.
- [6] ASTM Standard E45-97, "Standard Test Methods for Determining the Inclusion Content of Steel," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 157-170.
- [7] ASTM Standard E739-91, "Standard Practice for Statistical Analysis of Linear or Linearized Stress-Life (S-N) and Strain-Life (ε-N) Fatigue Data," Annual Book of ASTM Standards, Vol. 03.01, 1995, pp. 615-621.
- [8] ASTM Standard E646-93, "Standard Test Method for Tensile Strain-Hardening Exponents (n-values) of Metallic Sheet Materials," Annual Book of ASTM Standards, Vol. 03.01, 1997, pp. 550-556.
- [9] Bridgman, P. W., "Stress Distribution at the Neck of Tension Specimen," *Transactions of American Society for Metals*, Vol. 32, 1944, pp. 553-572.

APPENDIX

Specimen ID	D _{o,} mm (in.)	D _{f,} mm (in.)	L _{o,} mm (in.)	L _{f,} mm (in.)	E, GPa (ksi)	YS (offset=0 .2%), MPa (ksi)	UYS, MPa (ksi)	LYS, MPa (ksi)	YPE ,%	S _u , MPa (ksi)	K, MPa (ksi)	n	%EL, %	%RA, %	R, mm (in.)	σ _f *, MPa (ksi)	ε _f
F3-1	5.04	5.04	7.62	9.12	206.4	1412.5	NA	NA	NA	1677.5	2,720.3	0.1029	NA	NA	NA	NA	NA
	(0.198)	(0.198)	(0.30)	(0.36)	(29,935.0)	(204.9)				(243.3)	(394.5)						
F3-2	5.03	4.66	7.62	8.43	206.5	1390.8	NA	NA	NA	1676.9	2,327.8	0.0816	10.7%	14.2%	13.39	1851.7	15.3%
	(0.198)	(0.184)	(0.30)	(0.33)	(29,952.3)	(201.7)				(243.2)	(337.6)				(0.527)	(268.6)	
Average					206.5	1401.7				1677.2	2524.1	0.0923	10.7%	14.2%	13.39	1851.7	15.3%
values					(29,943.7)	(203.3)				(243.3)	(366.1)				0.527	(268.6)	

Table A.1: Summary of monotonic tensile test results

* The values of true fracture strength are corrected for necking according to the Bridgman correction factor.

	1 1			At midlifa (Nerry)								
					r	At midlife	(N 50%)	-		ł		
	Test	Test				$\Delta \epsilon_{\rm r}/2$	$\Delta \epsilon_{\rm p}/2$			2N50%,	[c]	
Specimen ID	control	freq	E, GPa	E', GPa	AE/2. %	(calculated	r (measure	$\Delta \sigma/2,$	σ _m , MPa	[a]	(2N _f) _{50%} , ¹⁵³	Failure
Specificities	mode	Hz	(ksi)	(ksi)	Lu 2, /.			MPa (ksi)	(ksi)	reversa	reversals	location [11]
	moue	112), %	u), 70			ls		
F3-8	strain	0.10	210.6	195.7	2.002%	1.131%	1.074%	1798.3	-40.9	2	2	IGL
			(30,550.0)	(28,381.0)				(260.8)	(-5.9)			
F3-17	strain	0.10	200.4	183.5	1.999%	1.179%	1.074%	1694.4	-36.4	4	8	IGL
			(29,060.0)	(26,609.6)				(245.7)	(-5.3)			
F3-18	strain	0.10	208.4	192.4	1.497%	0.748%	0.690%	1546.3	-32.4	16	38	IGL
			(30,230.0)	(27,903.3)				(224.3)	(-4.7)			
F3-4	strain	0.20	210.3	196.5	1.001%	0.378%	0.340%	1286.0	-21.0	256	522	IGL
			(30,500.0)	(28,500.2)				(186.5)	(-3.0)			
F3-12	splaceme	0.20			1.000%	0.378%		1283.9	56.0	200	414	IGL
								(186.2)	(8.1)			
F3-14	strain +	0.20	207.3	194.7	0.994%	0.360%	0.323%	1308.6	-24.2	256	722	IGL
d	lisplaceme	ent	(30,070.0)	(28,231.7)				(189.8)	(-3.5)			
F3-7	strain +	0.83	210.4	202.7	0.645%	0.107%	0.100%	1112.1	-2.9	2,032	6,986	IGL
	load	2.00	(30,520.0)	(29,393.8)				(161.3)	(-0.4)			
F3-9	strain +	0.83	207.1	200.8	0.647%	0.103%	0.089%	1122.5	-18.0	1,122	9,424	IGL
	load	2.000	(30,030.0)	(29,127.8)				(162.8)	(-2.6)			
F3-13	splaceme	0.83			0.650%	0.119%		1096.0	37.6	3,712	7,378	IGL
								(159.0)	(5.5)			
F3-5	strain +	2.00	210.4	207.2	0.499%	0.018%	0.021%	992.9	44.5	4,702	25,162	IGL
	load	5.0	(30,510.0)	(30,050.7)				(144.0)	(6.5)			
F3-6	strain +	2.0	206.9	205.3	0.499%	0.021%	0.021%	986.6	77.9	4,002	28,156	IGL
	load	5.0	(30,000.0)	(29,770.8)				(143.1)	(11.3)			
F3-16	strain +	2.0	207.5	207.0	0.500%	0.026%	0.023%	978.6	67.0	4,596	33,390	IGL
	load	5.0	(30,090.0)	(30,019.1)				(141.9)	(9.7)			
F3-10	strain +	2.0	208.4	209.6	0.400%	0.000%	0.000%	837.0	49.7	2,596	400,498	IGL[ss]
	load	5.0	(30,230.0)	(30,404.2)				(121.4)	(7.2)			
F3-11	load	10.0			0.400%			836.3	3.3		236,506	IGL
								(121.3)	(0.5)			
F3-15	load	10.0			0.400%			837.0	3.3		223,310	IGL[ss]
								(121.4)	(0.5)			
F3-19	load	22.0			0.325%			668.8	5.6		9,922,112	IGL[ss]
								(97.0)	(0.8)			
F3-20	load	25.0			0.325%			672.3	0.0		>10,000,000	No Failure
								(97.5)	(0.0)			
F3-21	load	25.0			0.325%			672.3	0.0		>10,000,000	No Failure
								(97.5)	(0.0)			

Table A.2: Summary of constant amplitude completely reversed fatigue test results

[a] $N_{50\%}$ is defined as the midlife cycle (for run-out tests, data is taken from the stable cycle indicated).

[b] $(N_f)_{50\%}$ is defined as 50% load drop.

[c] IGL = inside gage length.

[SS] Subsurface cracking (location as shown in the graph below with the hardness profile).



Figure A.1a: True stress amplitude versus normalized number of cycles



Figure A.1b: True stress amplitude versus number of cycles



Figure A.2: Composite plot of midlife hysteresis loops



Figure A.4: Monotonic stress-strain curves (stress-strain curve for core material was obtained from U. of Waterloo report)



Figure A.5: True stress amplitude versus true strain amplitude (stress-strain curve for core material was obtained from U. of Waterloo report)



Figure A.6: True stress amplitude versus reversals to failure (curve for core materials was obtained from U. of Waterloo report)



Figure A.7: Calculated true plastic strain amplitude versus reversals to failure (curve for core material was obtained from U. of Waterloo report)



Figure A.8: True strain amplitude versus reversals to failure (curve for core materials was obtained from U. of Waterloo report)



Figure A.9: Comparison of monotonic stress-strain curves for two 8620 composites



Figure A.10: Comparison of cyclic stress-strain curves for two 8620 composites



Figure A.11: Comparison curves of true strain amplitude versus reversals to failure for two 8620 composites



Figure A.12: Comparison curves of true stress amplitude versus reversals to failure for two 8620 composites



Figure a.13: Comparison curves of calculated true plastic strain amplitude versus reversals to failure for two 8620 composite 45



200 µm

Figure A.14: A typical subsurface failure at 50 X for SAE 8620 Case-Core Composite steel



Figure A.15 Comparison curves of residual stress profiles for two 8620 Composites



Figure A.16: Comparison curves of hardness profiles for two 8620 composites



Figure A.17a: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (2% strain amplitude)



Figure A.17b: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (1.5% strain amplitude)



Figure A.17c: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (1% strain amplitude)



Figure A.17d: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (0.65% strain amplitude)



Figure A.17e: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (0.5% strain amplitude)



Figure A.17f: Fracture surface of a surface failure for SAE 8620 Case-Core Composite steel (0.4% strain amplitude)



Figure A.17g: Fracture surface of a subsurface failure for SAE 8620 Case-Core Composite steel (0.4% strain amplitude)



Figure A.17f: Fracture surface of a subsurface failure for SAE 8620 Case-Core Composite steel (0.325% strain amplitude)