Weld Challenge III – Exhaust Hanger Weld Fatigue Problem

P. Dong and J.K. Hong/Battelle

P. Ramamohan, H. Agrawal/Ford Motor Company



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Fatigue Life Prediction Procedures

- Battelle's mesh-insensitive structural stress method
- Master S-N curve based on a large amount of MIG weld S-N data expressed in terms of

$$\Delta S_s = \frac{\Delta \sigma_s}{t^{[(2-m)/2m]} \cdot I(r)^{1/m}}$$

- FE Mesh was generated based on ArvinMeritor's CAD model from FD&E's website
- Assumptions/simplifications:
 - Failure criterion of the master S-N curve: through-thickness failure
 - Rob modeled using shell elements with t=r
 - Weld element representation: shell thickness = 3mm
 - I(r) for structural joint (i.e., load shielding effects):
 - SS analysis results: weld toe/end failure in pipe and weld metal failure (root or throat failures) not likely
 - Min. 6mm crack length in pipe



The Structural Stress Definition



Structural Stress: Equilibrium Equivalent



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Mesh-Insensitivity Demonstration – The Structural Stress Method



Master S-N Curve (Over 800 Tests): Load Controlled Conditions



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Model 1: A Simplified Weld Presentation to Check Potential Weld Failure - Unlikely



Weld represented by one row elements





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Model 2: Weld Toe Failure Into Pipe Wall

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Life Predictions: Weld Toe Failure Along the Left Weld Originating From End

Force amplitudes:1023N, 845N, 689NLoad ratio:R=-1





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Some Concluding Remarks

- Highly localized stress concentration at one weld end
 - Highly rigidity of rod
 - Flexible thin pipe wall
 - The specific loading mode
- Minimum 6mm crack length as failure criterion is too large to maintain load-controlled conditions
- The present mean predictions should be on the conservative side

